

**B.Tech II Year - II Semester Examinations, April/May-2012****THERMAL ENGINEERING - I  
(MECHANICAL ENGINEERING)****Time: 3 hours****Max. Marks: 80****Answer any five questions  
All questions carry equal marks**

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- 1.a) How do you classify the I C Engines based on governing mechanism, number of strokes and number of cylinders? Explain them.
- b) Draw the battery ignition system and explain the important points. [16]
- 2.a) What are different major losses involved in S I Engine in comparison with the air standard cycle and draw the actual cycle? Explain.
- b) What are different factors influencing the ignition lag of S I Engine? Explain them in detail. [16]
- 3.a) What is diesel knock? What are the favorable conditions for knocking in C I Engines? How to control knocking?
- b) Draw the schematic diagram of divided combustion chamber and explain its working. [16]
- 4.a) What are the good requirements for diesel fuel and explain the importance of fuel rating?
- b) Draw the actual and ideal P- $\theta$  diagram and explain different stages of combustion in S I Engine. [16]
- 5.a) How to measure the frictional power in I C Engine using retardation test? Explain.
- b) A power output (in kW) of six cylinder four stroke engine is absorbed by hydraulic dynamometer for which the law  $WN/20000$ , where W is Newton and the N is speed in rpm. The air consumption is measured by air box method. The following readings were obtained: orifice of diameter = 30 mm, stroke = 120mm, Bore = 100 mm, Brake load = 560 N, C/H ratio by mass 85/15,  $C_d$  of orifice = 0.6, Ambient pressure = 1 bar, Pressure drop across the orifice = 14.5 cm of Hg. Time taken for 100 cc fuel consumption = 20 sec. Ambient temperature =  $27^{\circ}\text{C}$ , Fuel density =  $835 \text{ kg/m}^3$ .  $N = 2400$ .  
Calculate  
i) Brake power            ii) Torque                    iii) BSFC  
iv) Percentage of excess air supplied            v) Volumetric efficiency. [16]
- 6.a) Derive the equation for the optimum pressure ratio for two stage compression with perfect inter cooling and extend it for n number of cylinders.
- b) A single stage double acting air compressor of 62.5 kW I.P. running at 120 rpm takes air at 1 bar and delivers at 10 bar. Assuming the law of expansion and compression as  $p v^{1.35} = \text{constant}$ , find the diameter and stroke of the cylinder if the stroke to bore ratio is 1.25. [16]

- 7.a) Differentiate between positive displacement compressors and dynamic compressors.
- b) A centrifugal air compressor delivers 15 kg of air, per minute. The inlet and outlet conditions of air are  $V_1 = 10$  m/s,  $p_1 = 1$  bar,  $v_{s1} = 0.5$  m<sup>3</sup>/kg, and  $V_2 = 80$  m/s,  $p_2 = 7$  bar,  $v_{s2} = 0.15$  m<sup>3</sup>/kg. The increase in enthalpy of air passing through the compressor is 160 kJ/kg and heat loss to the surroundings is 720 KJ/min. Find the power required to drive the compressor. Assume that inlet and discharge lines are at the same level. [16]
- 8.a) A refrigerating unit is working between 40<sup>0</sup>C and -10<sup>0</sup>C. The load on the unit is 5 tons. Find:
- i) COP of the system;
- ii) Power required to run system.
- Assume that the refrigerant is dry and saturated vapour leaving the evaporator and compression is isentropic. The refrigerant used is NH<sub>3</sub>.
- If the temperature of the refrigerant required in the evaporator is -20<sup>0</sup>C, then, find the change in COP of the system and the power required.
- b) Draw the layout of summer air conditioning system and explain its working principle along with psychrometric chart. [16]

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