

# 6.003 Homework 6

Due at the beginning of recitation on **Wednesday, March 17, 2010.**

## Problems

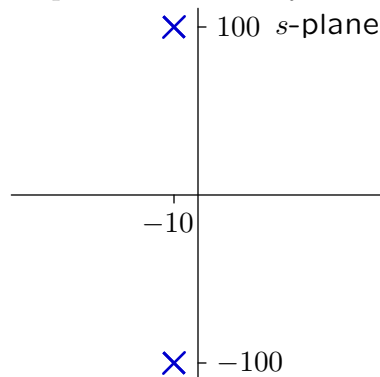
### 1. Second-order systems

The impulse response of a second-order CT system has the form

$$h(t) = e^{-\sigma t} \cos(\omega_d t + \phi) u(t)$$

where the parameters  $\sigma$ ,  $\omega_d$ , and  $\phi$  are related to the parameters of the characteristic polynomial for the system:  $s^2 + Bs + C$ .

- Determine expressions for  $\sigma$  and  $\omega_d$  (not  $\phi$ ) in terms of  $B$  and  $C$ .
- Determine
  - the time required for the envelope  $e^{-\sigma t}$  of  $h(t)$  to diminish by a factor of  $e$ ,
  - the period of the oscillations in  $h(t)$ , and
  - the number of periods of oscillation before  $h(t)$  diminishes by a factor of  $e$ .Express your results as functions of  $B$  and  $C$  only.
- Estimate the parameters in part b for a CT system with the following poles:



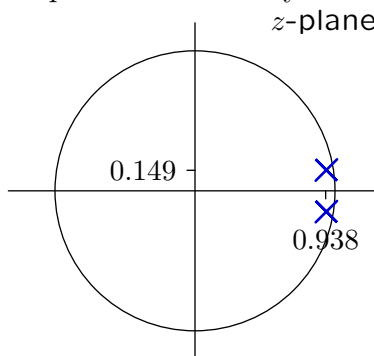
The unit-sample response of a second-order DT system has the form

$$h[n] = r_0^n \cos(\Omega_0 n + \Phi) u[n]$$

where the parameters  $r_0$ ,  $\Omega_0$ , and  $\Phi$  are related to the parameters of the characteristic polynomial for the system:  $z^2 + Dz + E$ .

- Determine expressions for  $r_0$  and  $\Omega_0$  (not  $\Phi$ ) in terms of  $D$  and  $E$ .
- Determine
  - the length of time required for the envelope  $r_0^n$  of  $h[n]$  to diminish by a factor of  $e$ .
  - the period of the oscillations in  $h[n]$ , and
  - the number of periods of oscillation in  $h[n]$  before it diminishes by a factor of  $e$ .Express your results as functions of  $D$  and  $E$  only.

- f. Estimate the parameters in part e for a DT system with the following poles:



## 2. Maximum gain

For each of the following systems, find the frequency  $\omega_m$  for which the magnitude of the gain is greatest.

a.  $\frac{1}{1 + s + s^2}$

b.  $\frac{s}{1 + s + s^2}$

c.  $\frac{s^2}{1 + s + s^2}$

Compare the  $\omega_m$  for these systems and explain qualitatively any similarities or differences.

## 3. Phase

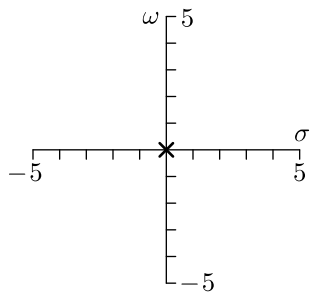
For a second-order system with poles at  $-1$  and  $-4$  (and no zeros), find the frequency at which the phase is  $-90^\circ$ , using any method except for the vector method. Then illustrate and confirm that result using the vector method.

## 4. Matches

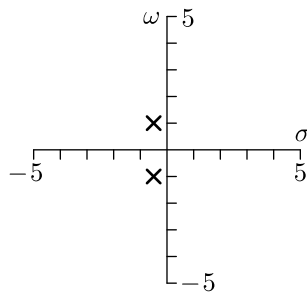
The following plots show pole-zero diagrams, impulse responses, Bode magnitude plots, and Bode angle plots for six causal CT LTI systems. Determine which corresponds to which and fill in the following table.

	$h(t)$	Magnitude	Angle
PZ diagram 1:			
PZ diagram 2:			
PZ diagram 3:			
PZ diagram 4:			
PZ diagram 5:			
PZ diagram 6:			

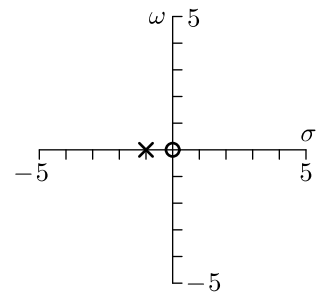
Pole-zero diagram 1



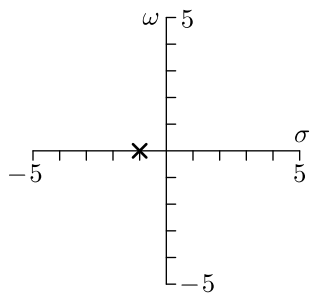
Pole-zero diagram 2



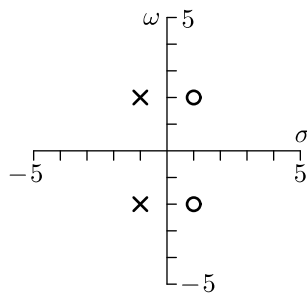
Pole-zero diagram 3



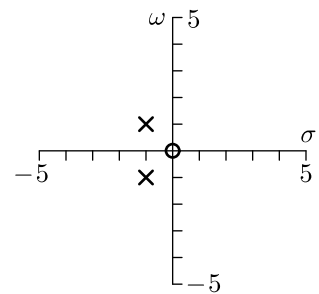
Pole-zero diagram 4



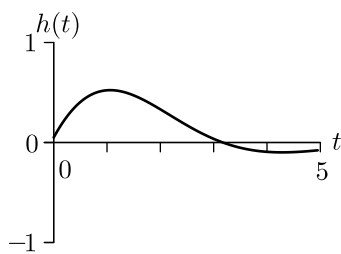
Pole-zero diagram 5



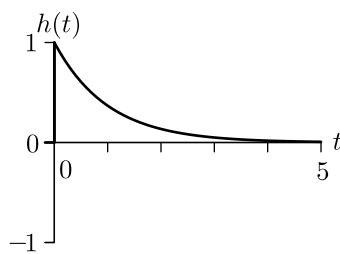
Pole-zero diagram 6



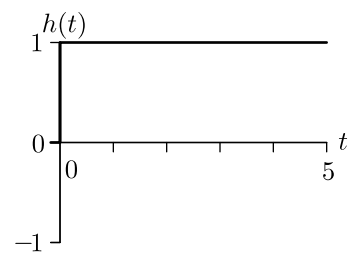
Impulse response 1



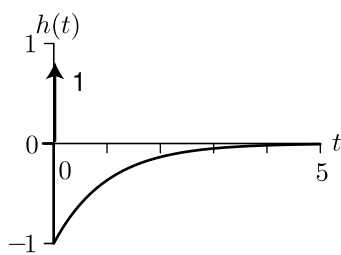
Impulse response 2



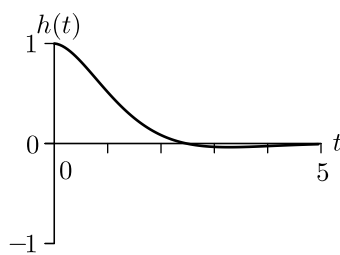
Impulse response 3



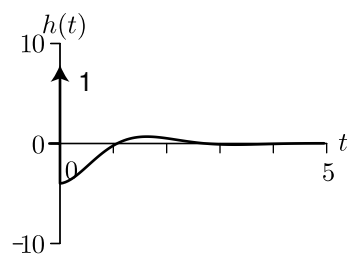
Impulse response 4



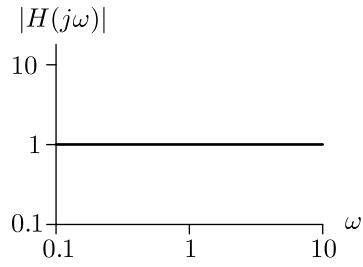
Impulse response 5



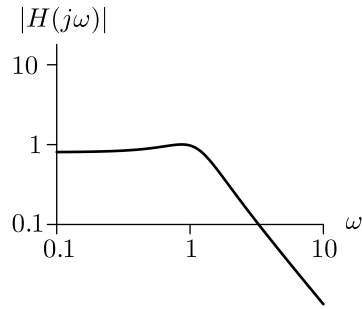
Impulse response 6



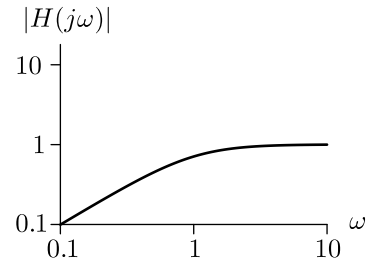
Bode Magnitude 1



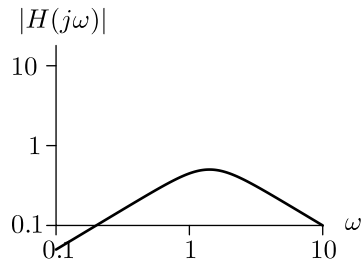
Bode Magnitude 2



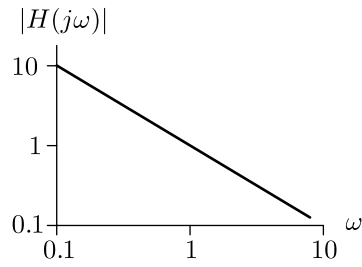
Bode Magnitude 3



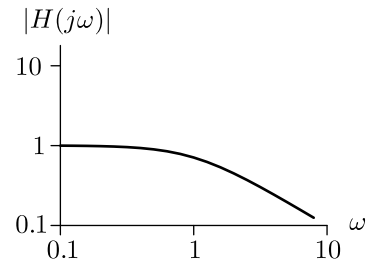
Bode Magnitude 4



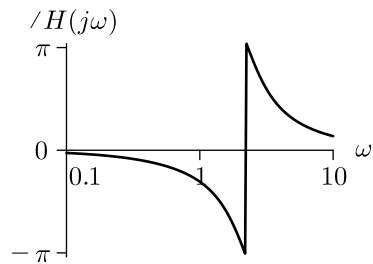
Bode Magnitude 5



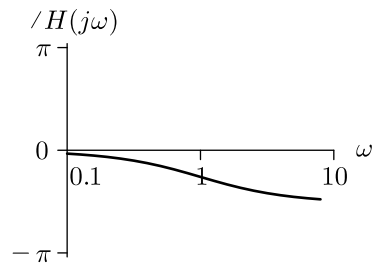
Bode Magnitude 6



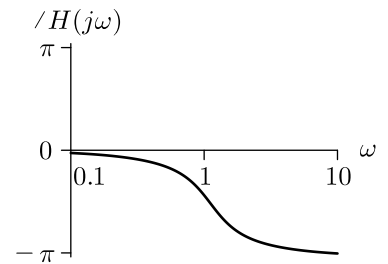
Bode Angle 1



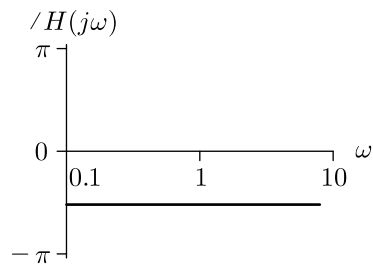
Bode Angle 2



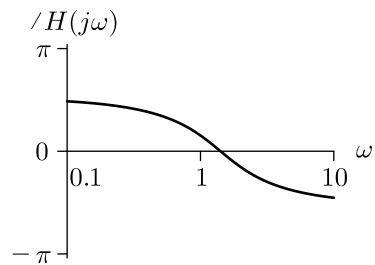
Bode Angle 3



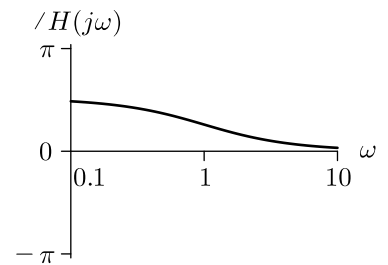
Bode Angle 4



Bode Angle 5



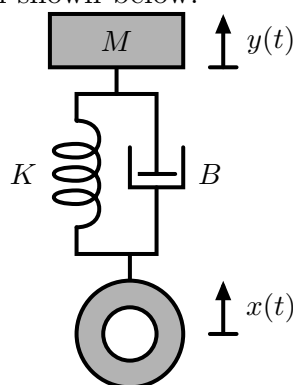
Bode Angle 6



## Engineering Design Problems

### 5. Automotive suspension

Wheels are attached to an automobile through a suspension system that is designed to minimize the vibrations of the passenger compartment that result when traveling over bumpy terrain. The suspension system consists of a spring and shock absorber that are both compressed when the wheel passes over a bump, so that the sudden motion of the wheel is not directly transmitted to the passenger compartment. The spring generates a force to hold the passenger compartment at a desired distance above the surface of the road, and the shock absorber adds frictional damping. In this problem, you will determine how much damping is desirable by analyzing a simple model of an automobile's suspension system shown below.



The model consists of a mass  $M$  that represents the mass of the car, which is connected through a spring and dashpot to the wheel. The vertical displacement of the wheel from its equilibrium position is taken as the input  $x(t)$ . The vertical displacement of the mass from its equilibrium position is taken as the output  $y(t)$ . The spring is assumed to obey Hooke's law, so that the force it generates is a constant  $K$  times the amount that the spring is compressed relative to its equilibrium compression. The shock absorber is assumed to generate a force that is a constant  $B$  times the velocity with which the shock absorber is compressed. Notice that by referring  $x(t)$  and  $y(t)$  to their equilibrium positions, the force due to gravity can be ignored. Assume that  $M = 1$  and  $K = 1$ .

- Determine the differential equation that relates the input  $x(t)$  and output  $y(t)$ .
- Determine and plot the impulse response of the system when  $B = 0$ . Based on this result, give a physical explanation of the problem that would result if there were no shock absorber in the system.
- Determine an expression for the smallest positive damping constant  $B$  for which the poles of the system have real values. Sketch the impulse response of the system for this value of  $B$ . Based on this result, give a physical explanation of how the shock absorber improves performance of the suspension system.
- Consider what would happen if  $B$  were very large. Sketch the impulse response for the system if  $B = 100$ . Describe how this response might be less desirable than that in part c. Provide a physical explanation for how a stiff shock absorber can degrade system performance.

## 6. Dial tones

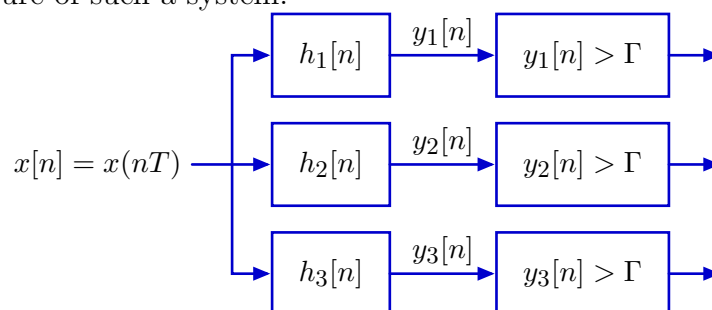
Pressing the buttons on a touch-tone phone generates tones that are used for dialing. Each button produces a pair of tones of the form

$$x(t) = \cos(2\pi f_1 t) + \cos(2\pi f_2 t)$$

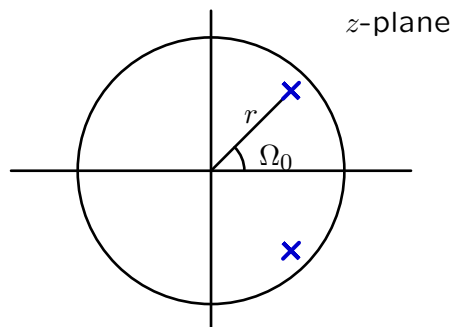
where  $f_1$  and  $f_2$  code the row and column of the button as shown in the following table.

$f_1$ [Hz]	$f_2$ [Hz]		
	1209	1336	1477
697	1	2	3
770	4	5	6
852	7	8	9
941	*	0	#

This problem concerns the design of a system to detect the row and column numbers that were pressed by analyzing the signal  $x(t)$ . The following block diagram illustrates the basic structure of such a system.



The input  $x(t)$  is first sampled with  $T = 10^{-4}$  seconds. The samples are then passed through LTI systems that generate intermediate signals so that  $y_1[n]$  is large when a button in column 1 is pressed,  $y_2[n]$  is large when a button in column 2 is pressed, and  $y_3[n]$  is large when a button in column 3 is pressed. These intermediate signals are then passed through detectors that determine when the signals are bigger than a threshold value  $\Gamma$ . Your task is to design the LTI systems. Each should consist of a system with 2 poles of the form shown in the following pole-zero diagram.



Such systems can be simulated by finding the difference equation that corresponds to the system and then iteratively solving that difference equation.

- a.** Determine values of  $r$  and  $\Omega_0$  so that the  $h_1[n]$  system generates a large response when the “1” key is pressed and a small response when the “2” or “3” keys are pressed. Your solution should work not only when the input consists of a single key press but also when it consists of sequences of key presses (as when dialing a phone number). Submit hardcopies of your code to generate  $y_1[n]$  along with a plot of  $y_1[n]$ .
- b.** Describe how the choice of  $\Omega_0$  affects the output signal  $y_1[n]$ .
- c.** Describe how the choice of  $r$  affects the output signal  $y_1[n]$ . In particular, what limits the maximum acceptable value of  $r$ ? Also, what limits the minimum acceptable value of  $r$ ?

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